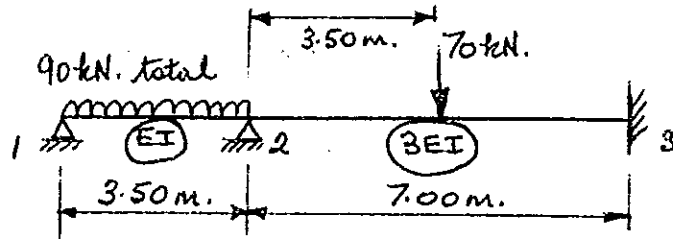


SLOPE DEFLECTION - Typical Examples.

Example. 1.

Determine the distribution of bending moment in the following two-span beam under the loading shown below.



F.E.M.s

$$\bullet \quad M_{23}^F = -M_{32}^F = -\frac{70 \times 7.00}{8} = -61.25 \text{ kN.m.}$$

$$M_{12}^F = -M_{21}^F = -\frac{90 \times 3.50}{12} = -26.25 \text{ kN.m.}$$

Slope-deflection equations for the ends of each member :-

$$M_{12} = M_{12}^F + S_{12}\theta + C_{21}S_{21}\theta_2 + \text{No chord rotation} \\ \text{ie } (S_{ij} + C_{ij}S_{ji})\phi = 0$$

$$= -26.25 + \frac{2EI}{3.50}(2\theta_1 + \theta_2) \quad \dots\dots\dots (1)$$

$$M_{21} = +26.25 + \frac{2EI}{3.50}(2\theta_2 + \theta_1) \quad \dots\dots\dots (2)$$

$$\bullet \quad M_{23} = -61.25 + \frac{2 \times 3EI}{7.00}(2\theta_2 + \theta_3) \quad \dots\dots\dots (3)$$

$$M_{32} = +61.25 + \frac{2 \times 3EI}{7.00}(2\theta_3 + \theta_2) \quad \dots\dots\dots (4)$$

We have 4 equations, and 7 unknowns. Derive the remaining equations from compatibility and equilibrium at the joints.

Compatibility $\theta_3 = 0 \quad \dots\dots (5)$

Equilibrium $M_{12} = 0 \quad \dots\dots (6)$

$$M_{21} + M_{23} = 0 \quad \dots\dots (7)$$

Unknowns are, $\theta_1, \theta_2, \theta_3, M_{12}$

M_{21}, M_{23}, M_{32}

\Rightarrow use compatibility

Hence substitute equations (2) and (3) into (7) to get

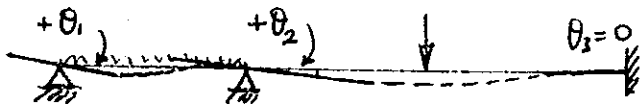
$$0.571 EI\theta_1 + 2.857 EI\theta_2 = 35.00 \quad \dots\dots (8)$$

Equate equation (1) to zero, from the equilibrium equation (6) we get

$$1.143 EI \theta_1 + 0.571 EI \theta_2 = 26.25 \dots (4)$$

Solving the simultaneous equations (8) and (9) gives

$$\theta_1 = + \frac{18.71}{EI} \quad ; \quad \theta_2 = + \frac{8.51}{EI}$$



Hence the joint (continuity) bending moments from equations (1) - (4) are

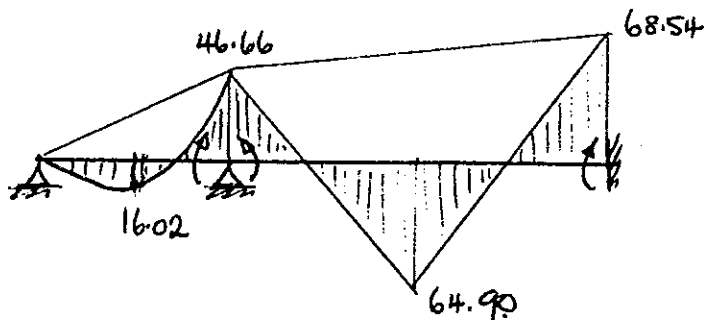
$$M_{12} = 0$$

$$M_{21} = +26.25 + \frac{2EI}{3.50} \left(2 \times \frac{8.51}{EI} + \frac{18.71}{EI} \right) = +46.66 \text{ kN.m.}$$

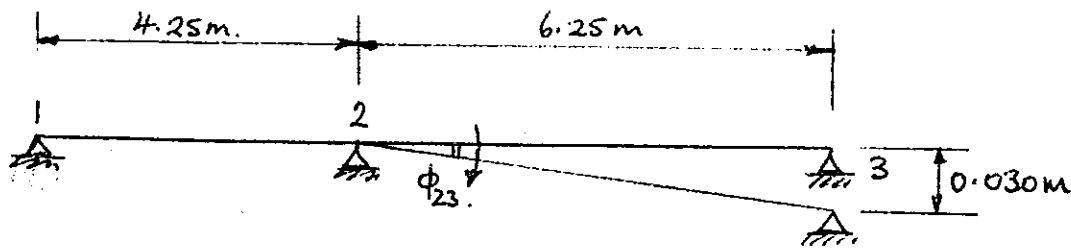
$$M_{23} = -61.25 + \frac{2 \times 3EI}{7.00} \left(2 \times \frac{8.51}{EI} + 0 \right) = -46.66 \text{ kN.m.}$$

$$M_{32} = +61.25 + \frac{2 \times 3EI}{7.00} \left(\frac{8.51}{EI} \right) = +68.54 \text{ kN.m.}$$

Bending moment diagram.



Determine the distribution of bending moments induced in the two-span beam when joint 3 settles by 30 mm. EI is constant along the full length of the beam.



The chord rotation ϕ_{23} is clockwise and positive, so the term $(\frac{3u}{L})$ is negative. The F.E.M. are all zero - no applied loads.

● Slope-deflection equations.

$$M_{12} = \frac{2EI}{4.25} (2\theta_1 + \theta_2) \text{ ----- (1)}$$

$$M_{21} = \frac{2EI}{4.25} (2\theta_2 + \theta_1) \text{ ----- (2)}$$

$$M_{23} = \frac{2EI}{6.25} (2\theta_2 + \theta_3 - \frac{3 \times 0.030}{6.25}) \text{ ----- (3)}$$

$$M_{32} = \frac{2EI}{6.25} (2\theta_3 + \theta_2 - \frac{3 \times 0.030}{6.25}) \text{ ----- (4)}$$

7 unknowns - 4 equations. Three further equations come from joint equilibrium.

$$M_{12} = 0 \text{ ----- (5)}$$

$$M_{21} + M_{23} = 0 \text{ ----- (6)}$$

$$M_{32} = 0 \text{ ----- (7)}$$

Solve :- From equations (1) and (5) $\theta_1 = -\frac{\theta_2}{2}$ ----- (8)

" " (4) " (7) $\theta_3 = 0.0072 - \frac{\theta_2}{2}$ ----- (9)

Substituting equations (2) and (3) into (6) gives

$$\frac{1}{4.25} (2\theta_2 + \theta_1) + \frac{1}{6.25} (2\theta_2 + \theta_3 - 0.0144) = 0 \text{ ----- (10)}$$

Using the relationships derived in (8) and (9) we now get

(6)

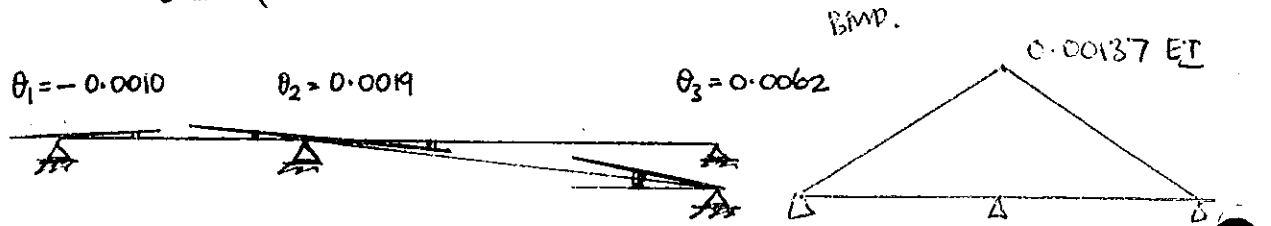
It follows that $\theta_1 = -0.00097$ radians.

and $\theta_3 = +0.0062$ radians.

Backsubstitute these values into the Slope-Deflection equation.

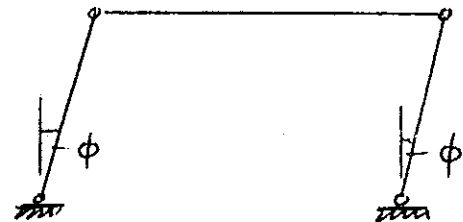
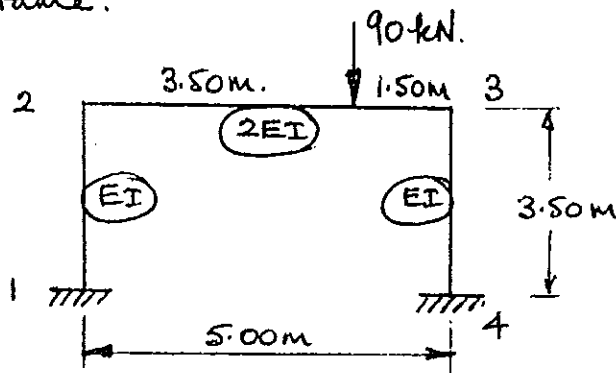
$$M_{21} = \frac{2EI}{4.25} (2 \times 0.00194 - 0.0010) = 0.00137 EI$$

$$M_{23} = \frac{2EI}{6.25} (2 \times 0.00194 + 0.0062 - 0.0144) = -0.00137 EI$$



Example 3.

Determine the distribution of bending moments in the following frame.



Sway Mechanism.

F.E.M. $M_{23}^F = -28.35 \text{ kN.m.}$

$$M_{32}^F = +66.15 \text{ kN.m.}$$

Slope-deflection equations.

$$M_{12} = \frac{2EI}{3.50} (2\theta_1 + \theta_2 - 3\phi) \text{ ----- (1)}$$

$$M_{21} = \frac{2EI}{3.50} (2\theta_2 + \theta_1 - 3\phi) \text{ ----- (2)}$$

$$M_{23} = \frac{2 \times 2EI}{5.00} (2\theta_2 + \theta_3) - 28.35 \text{ ----- (3)}$$

$$M_{32} = +66.15 + \frac{2 \times 2EI}{5.00} (2\theta_3 + \theta_2) \text{ ----- (4)}$$

$$M_{12}^F = M_{21}^F = 0$$

$$M_{43} = \frac{2EI}{3.50} (2\theta_3 + \theta_4 - 3\phi) \text{ ----- (5)}$$

$$M_{43} = \frac{2EI}{3.50} (2\theta_4 + \theta_3 - 3\phi) \text{ ----- (6)}$$

11 unknowns - 6 equations.

Compatibility $\theta_1 = 0$ ----- (7)

$\theta_4 = 0$ ----- (8)

Equilibrium $M_{21} + M_{23} = 0$ ----- (9)

$M_{32} + M_{34} = 0$ ----- (10)

One more equation is required namely the "Moment-Sway" equation of equilibrium.

$$M_{12} + M_{21} + M_{34} + M_{43} = 0 \text{ ----- (11)}$$

Solve:-

(a) Substitute equations (2) and (3) into equation (9), equaling $\theta_1 = 0$ (7)

$$2.743 \theta_2 + 0.8 EI \theta_3 - 1.714 \phi = 28.35/EI.$$

(b) Substitute equations (4) and (5) into equation (10), equaling $\theta_4 = 0$ (8)

$$0.8 \theta_2 + 2.743 \theta_3 - 1.714 \phi = -66.15/EI.$$

(c) Substitute equations (1), (2), (5) and (6) into equation (11)

$$\theta_2 + \theta_3 - 4\phi = 0.$$

In matrix form

$$\begin{bmatrix} +2.743 & +0.80 & -1.714 \\ +0.8 & +2.743 & -1.714 \\ +1 & +1 & -4 \end{bmatrix} \begin{Bmatrix} \theta_2 \\ \theta_3 \\ \phi \end{Bmatrix} = \begin{Bmatrix} 28.35/EI \\ -66.15/EI \\ 0 \end{Bmatrix}$$

Solve to get

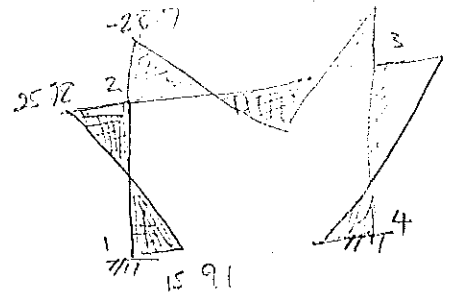
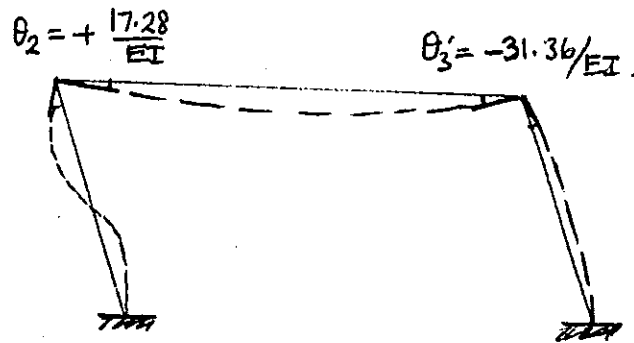
$$\theta_2 = \frac{17.28}{EI}$$

$$\theta_3 = -\frac{31.36}{EI}$$

$$+ \frac{3.52}{EI}$$

6.

Note that ϕ is negative so the frame sways in a direction opposite to the sway mechanism selected on page 4.



Backsubstitute to get the joint moments.

$$M_{12} = \frac{2EI}{3.5} \left(\frac{17.28}{EI} + 3 \times \frac{3.52}{EI} \right) = +15.91 \text{ kN.m.}$$

$$M_{21} = +25.78 \text{ kN.m.}$$

$$M_{23} = -25.78 \text{ kN.m.}$$

$$M_{32} = +29.81 \text{ kN.m.}$$

$$M_{34} = -29.80 \text{ kN.m.}$$

$$M_{43} = -11.89 \text{ kN.m.}$$

